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Wind turbine simulation model for the study of combined mechanical and electrical faults

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Abstract—Wind turbine (WT) is a complex system comprised of many different components with different functions; it is a formidable challenge to understand the dynamic behaviour of a WT and the root cause of any disturbances, owing to the integration of aerodynamic, mechanical and electrical components in an environment subject to significant stochastic environment forcing. Therefore, there is a need for robust and reliable model to study the performance of WTs in terms of truly integrated electrical and mechanical responses. This paper presents a new model for WT by using MATLAB. This model can be simulated under various operating conditions to yield valuable information for condition monitoring and effective algorithm development for fault detection.

Index Terms—Wind turbine, Combined simulation, MATLAB simulation, Fault detection.

I. INTRODUCTION

Wind power is one of the world’s fastest growing industries in recent years with countries striving to have more sustainable energy sources [1]. According to the global wind statistics of 2014, estimated by the global wind energy council (GWEC), wind power could reach 2,000 GW by 2030, and supply up to 17% -19% of global electricity by that time. By 2050, wind power could provide 25-30% of global electricity supply [2]. The European Wind Energy Association has envisioned that WT installations in the European Union will increase 64% by 2020 compared to 2013 levels [3]. Nevertheless, China foresees wind power capacity reaching 200 GW by 2020, 400 GW by 2030, and 1000 GW by 2050 [4].

As the large number of WTs are being installed and connected to power systems, it becomes more and more challenging for manufacturers to monitor and maintain WTs. Moreover, there is an additional cost might be caused, if WTs are not maintained timely. For example, failure of a $1500 bearing could result in a $100,000 gearbox replacement, a $50,000 generator rewind and another $70,000 in expenses to replace the failed components [5]. This can be extremely expensive especially considering that onshore turbines are generally in isolated areas, such as mountains, deserts and offshore [6].

To make wind energy as an attractive alternative option of power generation, it is necessary to reduce maintenance costs and improve the reliability of WTs. Better designs of WT components is one of the most considerable solutions for this problem; the other is condition monitoring system of WTs [7]. Condition monitoring system is required to detect and diagnose WT failures in their early stages. This will lead to reduce maintenance cost, hardware damaging and unscheduled downtime. With these aims in mind, guaranteeing robust and reliable model for WT is required to provide enough data for reliable monitoring.

Many researchers have developed dynamic models for WTs with some level of accuracy [8][9]. However, it is a formidable challenge to understand the dynamic behaviour of a wind turbine and the root cause of any disturbances, owing to the integration of both mechanical and electrical components in an environment subject to significant stochastic environment forcing. Part of the problem is probably due to mechanical engineers are only focusing on mechanical properties without considering the transient stability of the power system such as faults, voltage and frequency dips and unbalanced voltages. Similarly, it is difficult for electrical engineers to consider mechanical events such as (damaging torque variations from the drivetrain, relationships between angular velocity and torque, the impact of the gear ratios, gear stages, etc.). These often separate electrical and mechanical approaches may lead to inaccurate and unrealistic models.

In this paper, the purpose of the work is to present a comprehensive model in order to enable studies involving the integration of aerodynamic, mechanical and electrical properties of a WT. With these aims, the possibility of using current signals which measured from the terminals of the generator can be used for fault studies. A signal processing algorithm based on Fast Fourier Transform (FFT) is applied to monitor the amplitude of fault frequencies.

II. WIND TURBINE MODELING

MATLAB is a very valuable tool in many different contexts. It makes it possible to investigate the expected general behavior and performance of a WT during the steady state or the transient stability. The MATLAB environment is really a cost-effective method to perform very thorough and detailed investigations before a prototype is exposed to real full-scale tests. Consequently, the time and development costs can be reduced considerably when the concept and detail design can be thoroughly tested without exposing the physical prototypes to the inuence of destructive full-scale tests. The turbine modeled in this paper is a variable speed wind turbine based on a
multi-pole permanent magnet synchronous generator (PMSG). PMSG structure may or may not have a gearbox, and they are connected to the grid through a full-scale converter. But, using a gearbox would increase the speed of the generator shaft, and as a result, reduce the size of generator. The impact of the gearbox on the generator should, therefore, be investigated thoroughly in order to identify potential challenges and to develop measures to mitigate those issues. Accordingly, this work can provide an accurate and efficient way to analyze the interaction between mechanical system and electrical system in normal and fault operation. However, considering the modeling, WTs can be represented by a generic model as shown in Figure 1 [10].

Some components are not considered in this paper, including yaw systems, tower, bearings, brakes model and power converter. This is not considered a problem, because the main topic of interest in this work is to combine the aerodynamic, mechanical and electrical components in normal and fault operation, and the impact of those components on the stator current can be assumed to be rather limited. The subsystem models of WT are presented below.

A. Wind Speed Model

This model is used to generate a short-term wind speed variations with certain characteristics, such as speed range or turbulence intensity, which a WT will experience. The wind speed \( v_w \) is modeled as the sum of the four components [11].

\[
v_w(t) = v_{avg} + v_r(t) + v_g(t) + v_n(t) \tag{1}
\]

where \( v_{avg} \) is the average value of the wind speed, \( v_r(t) \) is the ramp wind component, \( v_g(t) \) is the gust wind component and \( v_n(t) \) is the base noise wind component, all of them in m/s. Figure 3 shows the graphics of the non-constant wind speed components: average, ramp, gust and noise components.

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**TABLE I**

<table>
<thead>
<tr>
<th>MODEL PARAMETERS.</th>
</tr>
</thead>
<tbody>
<tr>
<td><strong>Generator</strong></td>
</tr>
<tr>
<td>Line-Line Voltage (RMS)</td>
</tr>
<tr>
<td>Frequency</td>
</tr>
<tr>
<td>Stator phase resistance (ohm)</td>
</tr>
<tr>
<td>Armature inductance (H)</td>
</tr>
<tr>
<td>Inertia ( J(\text{kg.m}^2) )</td>
</tr>
<tr>
<td>Viscous damping ( F(\text{N.m.s}) )</td>
</tr>
<tr>
<td>Pole pairs</td>
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<tr>
<td><strong>Gearbox</strong></td>
</tr>
<tr>
<td>Gear Ratio</td>
</tr>
<tr>
<td>Input speed (RPM)</td>
</tr>
<tr>
<td>Output speed (RPM)</td>
</tr>
</tbody>
</table>

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**The model consists of the following subsystems:**

- Wind Speed Model;
- Rotor Model;
- Pitch Control System;
- Drivetrain Model;
- Generator model;
- Three-Phase RLC Load;

The schematic configuration used in this work is shown in Figure 2, and the important parameters used for this model are summarized in Table 1.
An example of a wind speed sequence generated using this approach is shown in Figure 4. The parameters used were \(v_{avg} = 12\), \(T_g = 90\), \(T_y = 250\), \(A_g = 1.5\), \(A_r = 2\), \(T_r = 400\), \(T_r = 500\).

![Wind Speed Sequence](image)

Fig. 4. Example of a simulated wind speed sequence

**B. Rotor Model**

The rotor transfers the kinetic energy from the wind into mechanical energy at the rotor shaft by aerodynamic forces producing lift on the blades. The kinetic energy of a cylinder of air of radius \(R\) traveling with wind speed \(v_w\) corresponds to a total wind power \(P_w\) within the rotor swept area of WT. This power, \(P_w\), can be expressed by

\[
P_w = \frac{1}{2} \rho \pi R^2 v_w^3
\]

where \(\rho\) is the air density (1.225 kg/m\(^3\)), \(R\) is the rotor radius and \(v_w\) is the wind speed. It is important to note that the rotor blades are designed to extract a maximum power from the wind, but it is not possible to convert all the air kinetic energy into electrical energy since the air would stand behind the turbine and it reduces the resulting wind speed. In other words, the wind speed is only reduced by the rotor blades, which thus extract a fraction of the power in the wind. This fraction is denominated the power efficiency coefficient \(C_P\) of the WT. The mechanical power \(P_m\) of the WT is therefore, by the definition of \(C_P\), given by the total power in the wind \(P_{wind}\) using the following equation:

\[
P_m = C_P P_w
\]

A WT has a maximum level of power which can be extracted from the wind so that the theoretical maximum energy that can be extracted is with \(C_P = 0.599\) and is termed the Betz limit. In practice the coefficient of performance is less than this and also depends on the specific WT shape, the wind velocity, the turbine rotor speed, and the turbines blade pitch, where the general function defining \(C_P\) as a function of the tip-speed ratio \(\lambda\) and the blade pitch angle \(\beta\) is defined as [12]

\[
C_P(\lambda, \beta) = c_1 \left( \frac{C_9}{\lambda_i} - c_3 \beta - c_4 \beta^2 - c_5 \right) \exp\left( -\frac{c_7}{\lambda_i} \right)
\]

where

\[
\lambda_i = \left( \frac{1}{\lambda + C_8 \beta} \right) - \left( \frac{C_9}{\beta^3 + 1} \right)^{-1}
\]

Equations (4) and (5) are used to calculate the impact of the pitch angle on the power coefficient. The resulting value can be inserted into Equation (2) to calculate the mechanical power extracted from the wind. The values of the constants can be found in [12].

If the mechanical torque \(T_m\) is to be applied instead of the mechanical power \(P_m\), it can be easily calculated from the dynamic theory of rotating devices in physics by using the turbine rotational speed \(\omega_m\):

\[
T_m = \frac{P_m}{\omega_m}
\]

where

\[
\omega_m = \frac{\lambda v_w}{R}
\]

Equations (2) and (6) are very important equations which combine the concepts of mechanical power, mechanical torque and total wind power.

**C. Pitch Control System**

The pitch control system is active only in high wind speeds above 12m/s. In these circumstances, the rotor speed can no longer be controlled by increasing the generated power, as this would lead to overloading the generator. Thus, the blade pitch angle is changed in order to limit the aerodynamic efficiency of the rotor. This prevents the rotor speed from becoming too high, which would result in mechanical damage. The flow chart shown in Figure 5 shows the different steps to calculate the pitch angle.

![Flow Chart of Pitch Control System](image)

Fig. 5. Flow chart of the pitch control system
The optimal pitch angle is zero below the nominal wind speed. From the nominal wind speed onwards, the optimal angle increases steadily with increasing wind speed. The performance of the model has been investigated under variable speed conditions in order to verify the performance of the pitch model. Figure 6 shows short-term wind speed variations of interest as input wind speed to the model. So, the mechanical power extracted from the wind is shown in Figure 7. It is clear that when the wind speed increases, the mechanical power increases until the pitching starts, when the speed reaches the rated speed, 12 m/s. While the blades pitch, the mechanical torque decreases.

\[
T_l = K \int (\omega_d - \omega_l) + B(\omega_d - \omega_l)dt \quad (8)
\]

where \(K\) (N.m.) is the shaft stiffness, \(B\) (N.m.s) is the internal damping, and \(\omega_d, \omega_l\) are the speeds (rad/s) of the driving side and the loaded side, respectively. While the mathematical model of the gearbox is described with the following equations:

\[
\omega_1 = N \omega_2 \\
T_2 = NT_1 \\
P_1 = \omega_1 T_1 \\
P_2 = -\omega_2 T_2 \quad (9)
\]

where \(\omega_1\) is input shaft angular velocity, \(\omega_2\) is the output shaft angular velocity, \(N\) is the gear ratio, \(T_1\) is the torque on the input shaft, \(T_2\) is the torque on the output shaft, and \(P_1, P_2\) are the power on the input shaft and the output shaft, respectively.

\[T_e = 1.5n_p[\psi_m i_{qs} + (L_d - L_q)i_{ds}i_{qs}] \quad (12)\]

\[
\frac{\partial}{\partial t} \omega_r = \frac{n_p}{J}(T_e - T_m) \quad (13)
\]

\[
\frac{\partial}{\partial t} \theta_r = \omega_r \quad (14)
\]

where \(d\) and \(q\) refer to the physical quantities that have been transformed into the \(d-q\) synchronous rotating reference frame, \(i_{ds}\) and \(i_{qs}\) are the stator currents on the \(d\) and \(q\) axis, respectively. \(R_s\) is the stator resistance, \(\psi_m\) is the permanent magnetic flux and \(n_p\) is the pole pairs. \(\omega_r\), \(\theta_r\), and \(J\) are the angular velocity of the rotor, the rotor angular and inertia of rotor, respectively.

\[\frac{\partial}{\partial t} i_{ds} = \frac{1}{L_d} v_{ds} - \frac{R_s}{L_d} i_{ds} + \frac{L_q}{L_d} \omega_r i_{qs} \quad (10)\]

\[\frac{\partial}{\partial t} i_{qs} = \frac{1}{L_q} v_{qs} - \frac{R_s}{L_q} i_{qs} - \frac{L_d}{L_q} \omega_r i_{ds} - \frac{\omega_r}{L_q} \psi_m \quad (11)\]

\[E. Generator model\]

Most modern large-scale variable speed WTs are based on doubly-fed induction generator (DFIG) and permanent magnet synchronous generator (PMSG). Therefore, PMSG is studied in this work. It must be noted that PMSG structure may or may not have a gearbox, and they are connected to the grid through a full-scale converter. But, using a gearbox would increase the speed of the generator shaft, and as a result, reduce the size of generator. However, the detail description and model equation derivation of PMSG can be found in most power system and electric machine references. The PMSG model can be described using following equations in the \(d-q\) rotor reference frame:

\[D. Drivetrain Model\]

The drive train consists of the following elements: a low speed shaft, a gearbox and a high speed shaft. The model outputs of the transmitted torque \(T_l\) through the shaft regarding the speed difference between the driving side and the loaded side of the shaft, is given by the following equation [13]:
III. SIMULATION RESULTS

In this section, the electrical responses to the generated wind speed are presented. The wind speed input used in these simulations is that shown in Figure 6, and the mechanical power extracted from the wind is shown in Figure 7. The WT rotor drives the generator through the gearbox that steps up the drive train rotational speed from about 20 rpm at the low-speed shaft to 1500 rpm at the high-speed shaft. As mentioned before, the model described enables studies involving the integration of aerodynamic, mechanical and electrical properties of a WT drive train. The model allows the study of the impact of high-ramping wind speed events on the power system, as well as the impact of turbulence on the voltage and frequency, and this can be seen on the electromagnetic torque and the stator currents of the generator in Figure 8 and Figure 9, respectively.

![Figure 8](image1.png)  
Fig. 8. PMSG electromagnetic torque and the stator current ($i_a$)

![Figure 9](image2.png)  
Fig. 9. PMSG stator currents ($i_b$) and ($i_c$)

The simulation results of the generator response show that the speed, however, contains an oscillatory component, which is rectified in the generator output, particularly the electromagnetic torque and the stator currents. Without any electrical control, the mechanical oscillations can be seen at the output.

![Figure 10](image3.png)  
Fig. 10. (a) Power spectrum of healthy WT. (b) Power spectrum of faulty WT.

Furthermore, the model is capable to simulate the transient responses of WT during different faults. For example, to verify the possibility to recognize useful features in generator signals when a mechanical fault occurs, a fault has been simulated by applying a friction force to change the low speed shaft during a simulation. The harmonic spectra of the healthy and faulty system was obtained by applying the fast fourier transform (FFT) algorithm to the stator current as shown in Figure 10.

![Figure 11](image4.png)  
Fig. 11. (a) Power spectrum of healthy WT. (b) Power spectrum of faulty WT.
In this work, another case study presented short circuits in the stator windings to ground which has been applied to see the characteristic and the shape of the stator current during the fault. Figure 11 shows that the stator winding fault can be well identified using the FFT algorithm. Compared to the previous case study, stronger oscillations of the current signal can be observed.

IV. Conclusion

In this work, a WT model is built in the MATLAB/Simulink environment. The model can simulate various operating conditions to study a WT in terms of its integrated aerodynamic, mechanical and electrical performance. The possibility of using current signals measured from the terminals of the generator is also investigated, and an FFT is applied to detect fault frequencies. The simulation results have shown that frequency components related to faults can be clearly recognized in the current spectrum. This model shows promising results and should provide an appropriate theoretical test bed for WT condition monitoring and effective algorithm development for fault detection, though further work is needed to model the different components of the WT in sufficient detail.

REFERENCES